

Chapter 4.5 Field Problems

Thermal problems (1dof – Temperature, T)

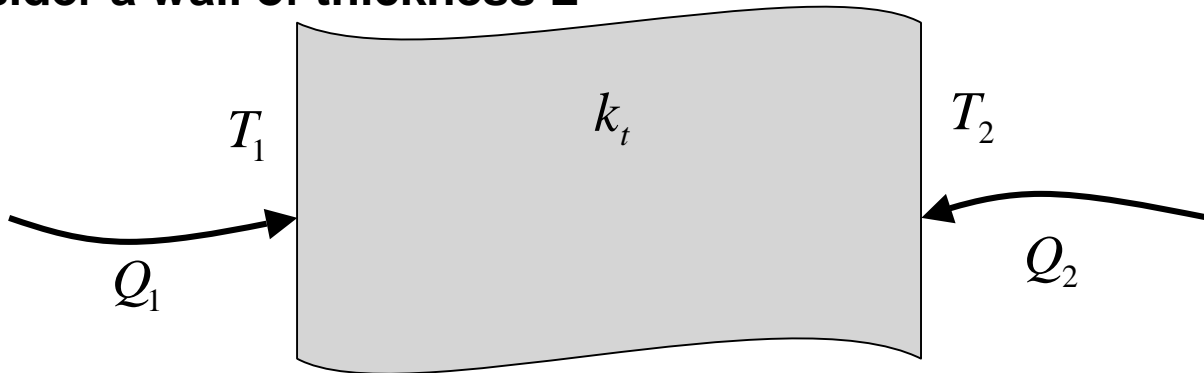
Electrostatic problems (1dof – Potential, V)

Antiplane problems (1dof – Out of plane displacement, w)

1-D Field Problems

(1-D heat conduction)

Consider a wall of thickness L

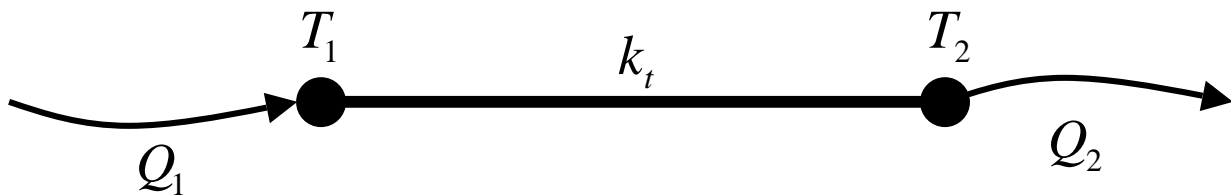


$$q = -k_t A \frac{dT}{dx}$$

$$q = Q_1 = -Q_2$$

$$\frac{dT}{dx} = \frac{T_2 - T_1}{L}$$

which we model as a single element



We have two equations with two unknowns T_1 and T_2

$$Q_1 = \frac{k_t A}{L} (T_1 - T_2)$$

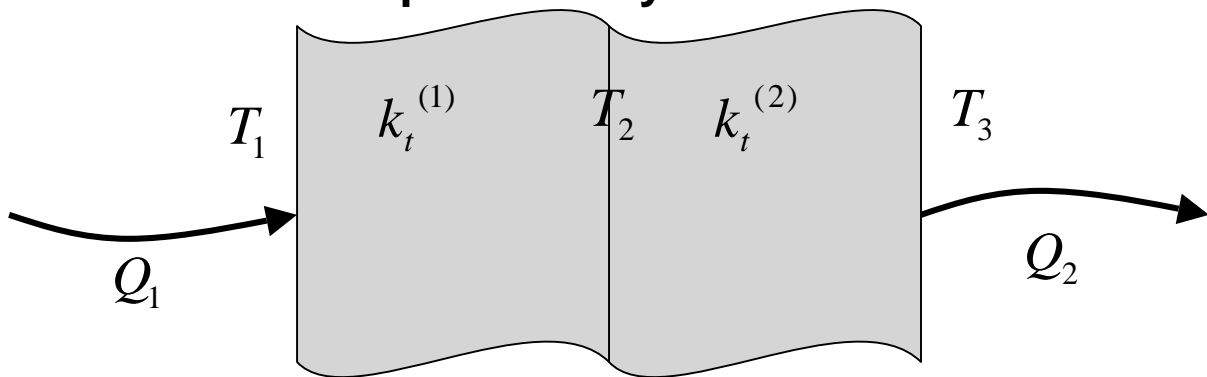
$$Q_2 = \frac{k_t A}{L} (T_2 - T_1)$$

We write this in matrix form as:

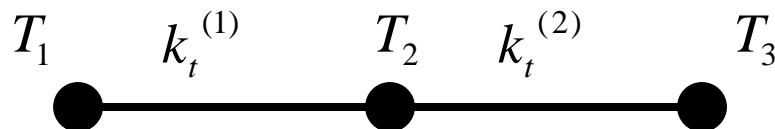
$$\begin{array}{c}
 \left\{ \begin{array}{l} Q_1 \\ Q_2 \end{array} \right\} = \begin{bmatrix} \frac{k_t A}{L} & -\frac{k_t A}{L} \\ -\frac{k_t A}{L} & \frac{k_t A}{L} \end{bmatrix} \left\{ \begin{array}{l} T_1 \\ T_2 \end{array} \right\} \\
 \text{Flux vector} \quad \quad \quad \text{Conductance matrix} \quad \quad \quad \text{Temperature vector}
 \end{array}$$

This is the same thing that we had for the 1-D truss element

Consider a wall made up of two layers



We model this as two elements



For element 1 we have

$$\left\{ \begin{array}{l} Q_1^{(1)} \\ Q_2^{(1)} \end{array} \right\} = \begin{bmatrix} \frac{k_t A^{(1)}}{L} & -\frac{k_t A^{(1)}}{L} \\ -\frac{k_t A^{(1)}}{L} & \frac{k_t A^{(1)}}{L} \end{bmatrix} \left\{ \begin{array}{l} T_1^{(1)} \\ T_2^{(1)} \end{array} \right\}$$

For element 2 we have

$$\left\{ \begin{array}{l} Q_2^{(2)} \\ Q_3^{(2)} \end{array} \right\} = \begin{bmatrix} \frac{k_t A^{(2)}}{L} & -\frac{k_t A^{(2)}}{L} \\ -\frac{k_t A^{(2)}}{L} & \frac{k_t A^{(2)}}{L} \end{bmatrix} \left\{ \begin{array}{l} T_2^{(2)} \\ T_3^{(2)} \end{array} \right\}$$

Now join the two elements

equilibrium requires that $Q_2^{(1)} + Q_2^{(2)} = Q_2$

compatibility requires that $T_2^{(1)} = T_2^{(2)} = T_2$

We really have the following 3 equations with 3 unknowns
In matrix from:

$$\begin{Bmatrix} Q_1 \\ Q_2 \\ Q_3 \end{Bmatrix} = \begin{bmatrix} \frac{k_{t1}A_1}{L_1} & -\frac{k_{t1}A_1}{L_1} & 0 \\ -\frac{k_{t1}A_1}{L_1} & \frac{k_{t1}A_1}{L_1} + \frac{k_{t2}A_2}{L_2} & -\frac{k_{t2}A_2}{L_2} \\ 0 & -\frac{k_{t2}A_2}{L_2} & \frac{k_{t2}A_2}{L_2} \end{bmatrix} \begin{Bmatrix} T_1 \\ T_2 \\ T_3 \end{Bmatrix}$$

Or

$$\mathbf{q} = \mathbf{Kt}$$

Nt. We don't have to rotate this $-T$ has no direction

2-D Field Problems

(General case)

The field variable is ϕ

The steady state problem is given by

$$\frac{\partial}{\partial x} \left(k_{xx} \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left(k_{yy} \frac{\partial T}{\partial y} \right) + Q = 0$$

Which, when $k = \text{constant}$, reduces to the well known Poisson's equation as:

$$\nabla^2 \phi = \frac{Q}{k}$$

Which, when $Q = 0$, reduces to the well known Laplace equation as:

$$\nabla^2 \phi = 0$$

For now let's consider the more general case

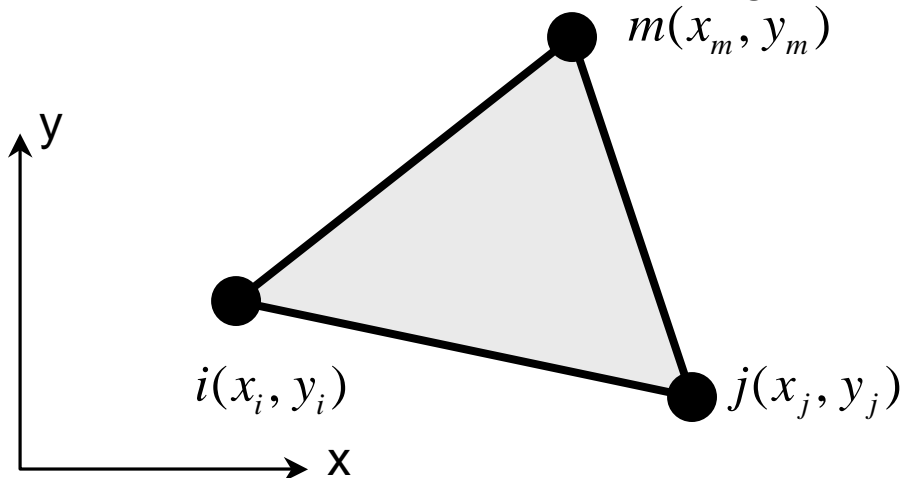
$$\frac{\partial}{\partial x} \left(k_{xx} \frac{\partial T}{\partial x} \right) + \frac{\partial}{\partial y} \left(k_{yy} \frac{\partial T}{\partial y} \right) + Q = 0$$

2-D heat transfer finite element derivation

Again we are going to go through the same steps as before

Step 1 Select element type

Consider the three noded triangular element



Nt. Node numbering is clockwise

Here each node has only 1 dof (Temperature)

Step 2 Select a temperature function

Since we have 3 nodes and 1 dof per node, let

$$T(x, y) = a_1 + a_2x + a_3y$$

which upon solving for the a's can be written as:

$$T(x, y) = \begin{bmatrix} N_1 & N_2 & N_3 \end{bmatrix} \begin{Bmatrix} T_i \\ T_j \\ T_m \end{Bmatrix} = \mathbf{Nt}$$

where the shape functions are solved as follows:

We really have 3 equations here

In matrix form we have for u_x

$$\begin{Bmatrix} T_i \\ T_j \\ T_m \end{Bmatrix} = \begin{bmatrix} 1 & x_i & y_i \\ 1 & x_j & y_j \\ 1 & x_m & y_m \end{bmatrix} \begin{Bmatrix} a_1 \\ a_2 \\ a_3 \end{Bmatrix}$$

Which can be solved as:

$$\mathbf{a} = \mathbf{x}^{-1}\mathbf{t}$$

To get the inverse of \mathbf{x} we use the method of cofactors

$$\mathbf{x}^{-1} = \frac{1}{2A} \mathit{adj}[\mathbf{x}]$$

where

$$2A = \det \begin{bmatrix} 1 & x_i & y_i \\ 1 & x_j & y_j \\ 1 & x_m & y_m \end{bmatrix}$$

A is the area of the triangle and

$$\mathit{adj}[\mathbf{x}] = (\mathit{cof}[\mathbf{x}])^T$$

Step 3 Define the temperature gradient/temperature and heat flux/temperature gradient relations

$$\mathbf{g} = \begin{Bmatrix} \frac{\partial T}{\partial x} \\ \frac{\partial T}{\partial y} \end{Bmatrix} = \mathbf{B}\mathbf{t}$$

where \mathbf{g} is the temperature gradient matrix (analogous to strain) and

$$\mathbf{B} = \frac{1}{2A} \begin{bmatrix} \beta_i & \beta_j & \beta_m \\ \gamma_i & \gamma_j & \gamma_m \end{bmatrix}$$

The heat flux/temperature gradient relation is:

$$\mathbf{q} = \begin{Bmatrix} q_x \\ q_y \end{Bmatrix} = \mathbf{D}\mathbf{g} = \begin{bmatrix} k_{xx} & 0 \\ 0 & k_{yy} \end{bmatrix} \begin{Bmatrix} \frac{\partial T}{\partial x} \\ \frac{\partial T}{\partial y} \end{Bmatrix}$$

Step 4 Derive the element conduction matrix and equations

Here we are going to use the variational method (analogous to the principle of minimum potential energy seen earlier)

Given the functional

$$\pi_h = U + \Omega_Q + \Omega_q + \Omega_h$$

Internal Energy

$$U = \iiint_V \frac{1}{2} \left[k_{xx} \left(\frac{\partial T}{\partial x} \right)^2 + k_{yy} \left(\frac{\partial T}{\partial y} \right)^2 \right] dV$$

Potential energy of the external forces

$$\Omega_Q = -\iiint_V QT dV$$

$$\Omega_q = -\iint_{S_q} q^* T ds_q$$

$$\Omega_h = \iint_{S_h} \frac{1}{2} h(T - T_\infty)^2 ds_h$$

Where Q is internal heat generation, q is heat flow and h is convection loss

(Nt, we can't specify both q and h on the same surface)

We can rewrite in matrix form as:

$$\pi_h = \iiint_V \frac{1}{2} \mathbf{g}^T \mathbf{D} \mathbf{g} \, dv - \iiint_V \mathbf{t}^T \mathbf{N}^T Q \, dV$$

$$- \iint_{S_q} \mathbf{t}^T \mathbf{N}^T q^* \, ds_q + \iint_{S_h} \frac{1}{2} h (\mathbf{t}^T \mathbf{N}^T - T_\infty)^2 \, ds_h$$

\mathbf{t} is not a function of x and y so lets pull it out of the integrals to get

$$\pi_h = \frac{1}{2} \mathbf{t}^T \iiint_V \frac{1}{2} \mathbf{B}^T \mathbf{D} \mathbf{B} \, dv \, \mathbf{t} - \mathbf{t}^T \iiint_V \mathbf{N}^T Q \, dV$$

$$- \mathbf{t}^T \iint_{S_q} \mathbf{N}^T q^* \, ds_q$$

$$+ \iint_{S_h} \frac{1}{2} h (\mathbf{t}^T \mathbf{N}^T \mathbf{N} \mathbf{t} - \mathbf{t}^T \mathbf{N}^T T_\infty + T_\infty^2) \, ds_h$$

We want to find the minimum energy (differentiate w.r.t. \mathbf{t} and = 0)

$$\frac{\partial \pi_h}{\partial \mathbf{t}} = \iiint_V \mathbf{B}^T \mathbf{D} \mathbf{B} \, dv \, \mathbf{t} - \iiint_V \mathbf{N}^T Q \, dV$$

$$- \iint_{S_q} \mathbf{N}^T q^* \, ds_q$$

$$+ \iint_{S_h} h (\mathbf{N}^T \mathbf{N} \mathbf{t}) \, ds_h + \iint_{S_h} \frac{1}{2} h (\mathbf{N}^T T_\infty) \, ds_h$$

And rewriting

$$\left[\iiint_V \mathbf{B}^T \mathbf{D} \mathbf{B} \, dv + \iint_{S_h} h(\mathbf{N}^T \mathbf{N}) \, ds_h \right] \mathbf{t} =$$

$$+ \iiint_V \mathbf{N}^T Q \, dV + \iint_{S_q} \mathbf{N}^T q^* \, ds_q - \iint_{S_h} \frac{1}{2} h(\mathbf{N}^T T_\infty) \, ds_h$$

And Voila!!

$$\mathbf{K} \mathbf{t} - \mathbf{f} = 0$$

We can simplify the conduction matrix since all terms in the \mathbf{B} matrix are constant as

$$\mathbf{K} = \left[\iiint_V \mathbf{B}^T \mathbf{D} \mathbf{B} \, dv + \iint_{S_h} h(\mathbf{N}^T \mathbf{N}) \, ds_h \right]$$

$$= t \mathbf{A} \mathbf{B}^T \mathbf{D} \mathbf{B} + h \iint_{S_h} (\mathbf{N}^T \mathbf{N}) \, ds_h$$

Step 5 Assembly

Use the same approach as shown before by applying compatibility and equilibrium

Step 6 Solve for the nodal temperatures...

Step 7 Postprocess for the temperature gradients and heat flux